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Realization of compact tractor beams using acoustic delay-lines
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Realization of compact tractor beams using acoustic delay-lines

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A method for generating stable ultrasonic matter in air using single beams (also known as tractor beams) is demonstrated. The method encodes the required phase modulation in passive unit cells into which the ultrasonic sources are mounted. These unit cells use waveguides such as straight and coiled tubes to act as delay-lines. It is shown that a static tractor beam can be generated using a single electrical driving signal, and a tractor beam with one-dimensional movement along the propagation direction can be created with two signals. Acoustic tractor beams capable of holding millimeter-sized polymer particles of density 1.25 g/cm³ and fruit-flies (Drosophila) are demonstrated. Based on these design concepts, we show that portable tractor beams can be constructed with simple components that are readily available and easily assembled, enabling applications in industrial contactless manipulation and biophysics. © 2017 Author(s). All article content, except where otherwise noted, is licensed under a Creative Commons Attribution (CC BY) license (http://creativecommons.org/licenses/by/4.0/). [http://dx.doi.org/10.1063/1.4972407]

Sound is a mechanical wave and as such it carries momentum that can generate acoustic radiation forces. When these forces are strong enough and converge from all directions, particles can be levitated against gravity. Acoustic levitation is becoming a fundamental tool in lab-on-a-chip scenarios, microscopy, pharmaceuticals, the levitation of biological samples and even small animals.

Recently, single-beam acoustic levitators have been generated using phased arrays. We adopt terminology common in optics and term these single-beams that can trap objects in three dimensions tractor beams. Acoustic tractor beams have the potential to revolutionize contactless manipulation due to their high exerted force to input power ratio and the wide variety of supported materials as well as sizes. However, phased array systems are currently necessary in order to generate and amplify dozens of independent electric signals making tractor beams not generally accessible due to cost, space, or complexity of operation.

Phased arrays are a collection of elements that emit or receive with specific phases or time delays. They are in widespread use in radar, sonar and ultrasonic imaging since they can dynamically change the direction or shape of the beam. At present, the phase delays are programmable with passive elements. For instance, electrical signals were made to travel through long cables to introduce a time delay proportional to its length. Inspired by these early passive delay lines, we developed devices that modulate a simple wavefront in order to generate an acoustic tractor beam. The sonic devices are composed of several unit cells, and the emitters are mounted into them. Each unit cell introduces a specific phase delay. Our approach parallels developments in 2D planar metamaterials where a layer of sub-wavelength unit-cells is used to shape and steer incident fields. In our sonic devices, the phase-shifting unit cells are not limited to the size of the wavelength and the emitters are designed as part of the structure. This approach minimises operational cost and complexity, making tractor beams portable and more readily deployable, hence enabling applications in industrial contactless manipulation and biophysics.

The acoustic pressure $P$ at point $r$ due to a collection of $N$ transducers emitting through a layer of passive delay lines can be written as

$$P(r) = \sum_{n=1}^{N} P_n L_n \frac{D_f(\theta_n)}{d_n} e^{i(\phi_n + k\Delta_n)},$$

where $P_n$ is a constant that defines the transducer output power, $L_n$ is a loss factor associated with each delay line, and $D_f$ is a far-field directivity function that depends on the angle $\theta_n$ between the effective source normal and $r$. Here, $D_f = 2J_1(ka \sin \theta_n)/ka \sin \theta_n$, the directivity function of a piston source, where $J_1$ is the first order Bessel function of the first kind and $a$ is the radius of the piston. The term $1/d_n$ accounts for divergence, where $d_n$ is the propagation distance in free space. The phase of the wave at point $r$ is $\phi_n + k\Delta_n$, where $\phi_n$ is the applied electronic phase, $\Delta_n$ is the propagation distance between $r$ and the emitters, $k = 2\pi/\lambda$ is the wavenumber, and $\lambda$ is the wavelength.

It should be noted that the phase of a wave arriving at $r$ can be controlled either by the electronic phase modulation $\phi_n$ or by the distance that the wave travels $\Delta_n$. Fig. 1 (Multimedia view) illustrates the three concepts that we used to code the phases of an electronic phased array into passive sonic devices. In the electronic phased array, the circuitry controls $\phi_n$ whereas in the passive devices $\phi_n$ is a constant for all the elements and the phase is controlled by changing $\Delta_n$.

**Electronic phased array.** An electronic phased array in which the elements are all located in a horizontal plane ($z = 0$) is shown in Fig. 1(a). Here, $\Delta_n = d_n = |r - s_n|$ and

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three distinct phase signatures have been shown to generate an array at a target position and then adding a static phase shift within the delay lines by lengthening or shortening them by a distance of \( \lambda / 2 \) or by reversing the polarity of the emitters. Introducing the signatures directly to the delay lines creates an integrated simpler structure but introduces small dissimilarities in the attenuation per element as different length tubes or coils are required. Switching the polarity requires a specific cable-transducer attachment and so is not such a flexible solution, but it leads to simple and more uniform surfaces. For the Straight tubes and Coils, we used the former method whereas for the Sculpted surface we used the latter leading to a simpler surface with all the emitters having the same distance to the target.

For the practical realization of the devices, we used transducers operating at 40 kHz (\( \lambda = 8.5 \text{ mm} \)): 10 mm diameter transducers (MA40S4S) for the Straight Tubes and Coils, and 16 mm diameter transducers for the Sculpted Surface (MCUSD16P40B12RO). With this configuration, the Straight tubes had a directivity (\( D_f \)) similar to that of the piston source (see supplementary Figs. 1 and 2), the linear relation (\( h = \varphi / k \)) was found to hold (\( R^2 = 0.997 \)) (see supplementary Fig. 3), and the loss factor was \( L_\text{loss} = 0.8 \) on average (see supplementary Fig. 4). For the coils, the relation between coil revolutions and phase modulation is presented in supplementary Fig. 5. The coils were 10 mm diameter at the entrance and exit and 2 mm in the inner region; this configuration reduced the insertion loss (see supplementary Fig. 6). The coil horns produced a directivity function similar to a piston source (see supplementary Figs. 1 and 2). The insertion loss for the coils depending on the revolutions and on average was \( L_\text{loss} = 0.7 \) (see supplementary Fig. 7). The devices developed to create a Twin trap using a single driving signal are illustrated in Fig. 2 (Multimedia view). These devices are portable as well as created with components that can be bought and put together by anyone (A step by step video is included in the supplementary material, and the source code and 3D files can be found online).

When the three passive devices are powered with a single driving signal, they generate a tractor beam. In Figs. 3(a)–3(d) (Multimedia view), we show the modelled and experimental acoustic pressure amplitude (see supplementary material) for a Twin trap generated with a Coil device. Figs. 3(e)–3(h)
show a Vortex trap generated with a Straight tubes device. The agreement between experiment and prediction is good, especially around the trapping point. Also, full three-dimensional Finite-Difference Time-Domain (FDTD) simulations were performed and found to be in good agreement with the simple model expressed by Eq. (1) (see supplementary Fig. 8).

The three devices were able to levitate expanded poly- styrene (EPS) spheres of $R_p = 2$ mm, which given a density $\rho_{\text{EPS}} = 29$ kg/m$^3$ represents a levitation force of 1 $\mu$N. The size limit is imposed as the devices levitate in the Rayleigh scattering regime in which the particle has to be small in comparison to the wavelength. The EPS particles were levitated vertically (i.e., above an array located in the x-y plane) and laterally (i.e., to the side of an array located in the x-z plane); in the case of the Sculpted surface devices also levitation upside down was achieved (i.e., below an array located in the x-y plane). Additionally, it was possible to levitate a particle of polymer (Polyactic Acid, $\rho_{\text{PLA}} = 1250$ kg/m$^3$, dia. 1.85 mm, thickness 0.5 mm) as well as fruit-flies (Drosophila, 0.3 mg each). A collection of these scenarios is shown in Fig. 4 (Multimedia view).

Device performance was measured experimentally by levitating a $R_p = 1$ mm EPS sphere with a Twin trap 1.8 cm above the surface and finding the minimum required voltage. This ranked the Sculpted surface as the most effective device, followed by the Straight tubes and then the Coil device. In brief, a drive signal of 14Vpp (5.6 W) was required for levitation using the Sculpted surface, 34Vpp (27.2 W) for the Straight tubes, and 43Vpp (51.6 W) for the Coils. In the normal regime of operation, the pressure generated by an emitter is proportional to the voltage of the driving signal. The trapping stiffness is the Laplacian of the Gor’kov Potential, which is proportional to the square of the pressure. Therefore, the trapping stiffness generated by our devices can be expressed as $K = \varepsilon V^2$, where $\varepsilon$ is a coefficient that defines the device efficiency and $V$ is the applied voltage. The $\varepsilon$ was 2.35 for the sculpted surface, 0.39 for the Straight tubes, and 0.24 for the Coils. These differences can be explained by the relatively high loss factors of the Coils (on average $L_n = 0.7$) and because the flat arrangement of transducers incurs losses due to directivity effects (i.e., the transducers are not directed to the focal point). Good agreement was found with Eq. (1) (RMSD = 0.06), which gave
predicted stiffness coefficients of $\varepsilon = 2.25, 0.39$ and 0.31, respectively.

The three devices were excited using a single electrical drive signal and consequently they produced a static trap. However, it is possible to exploit geometric symmetries to achieve dynamic focusing of the trap using a reduced number of signals. We first note that in an axisymmetric arrangement the distance between any point on the revolution axis and the transducers in a given row is the same; therefore, a single signal can be used to drive all the transducers in that row, swapping the polarity in the opposite half of that row to get a Twin trap. We then achieve dynamic trapping by following the technique used in concentric-ring phased arrays to refocus the beam at different heights. We used the Sculpted Surface device presented in Fig. 2(b) with the bottom row of transducers removed, leaving only the top and middle rows; the electronic circuit is presented in supplementary Fig. 9 and the arrangement in supplementary Fig. 10. To explore the functionality of this two channel arrangement, the particle was placed in the natural focus of the bowl (1.8 cm above the top) and the relative phase of the two 25Vpp excitation signals was changed to move the particle upwards in increments of 0.5 mm until the particle fell out of the trap under gravity. With this setup, the maximum height reached with an EPS bead $R_p = 1$ mm was 4.5 cm above the top of the spherical cap. Axisymmetric designs of the other sonic device concepts would enable them to use this feature.

Three types of passive devices capable of generating acoustic tractor beams have been demonstrated. They employ a small number of electric driving signals, making them a significant simplification over existing tractor beams which require electronic phased arrays with dozens of independent signals. The Sculpted surface sonic device is the most efficient device but Straight tubes and Coils have the advantage of presenting a flat emitting surface, making them more widely deployable. Additionally, Coil devices could use a single source since they are also flat on the lower surface. The ability of phased arrays to steer the beam has been replaced here in favour of a more portable, affordable, simple, and compact solution. This trade-off is desirable for scenarios that require levitation at fixed locations (which could potentially be coupled with mechanical motion) or limited movement with the addition of a small number of independent channels. The presented sonic devices have the potential to make acoustic tractor beams a widespread solution in contactless manipulation.

See supplementary material for information about the construction of the sonic devices, the driver board, as well as the simulated and experimental fields. It also contains experimental measures and model fits for the directivity of each delay line (supplementary Figures 1 and 2), phase modulation for the tubes (3), insertion loss of tubes (4), phase modulation for different coil revolutions (5), attenuation of different horns (6), insertion loss for different coil revolutions (7), FDTD simulations (8), Electronic circuit (9), and dynamic refocusing (10). The clip “Dynamic Trap” shows a simulation of the trap being refocused dynamically using 2 channels. The clip “Step by Step assembly” shows how to assemble a portable tractor beam step by step.

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FIG. 4. Different particles levitated in the three sonic devices. (a) and (b) Coil device, (c) and (d) Straight tube device, and (e)–(h) Sculpted surface device. (a) Four $R_p = 1$ mm EPS beads levitating 1.8 cm above a Coil device and (b) a single one levitating 2.5 cm above. (c) Three $R_p = 1$ mm beads levitated in a Straight tube device and (d) a $R_p = 2$ mm bead with three $R_p = 1$ mm on top. (e) Sculpted surface device levitating two $R_p = 1$ mm beads, (f) $R_p = 2$ mm bead upside down, (g) two fruit-flies levitated vertically, and (h) PLA particle levitating below an $R_p = 2$ mm EPS bead. (Multimedia view) [URL: http://dx.doi.org/10.1063/1.4972407.4]
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